

$$x'(t) = F(x, y)$$

$$y'(t) = G(x, y)$$

Let $x(t) \equiv x_*, y(t) \equiv y_*$ be an equilibrium solution, i.e.

$$F(x_*, y_*) = 0$$

 $G(x_*, y_*) = 0$.

For solutions $[x(t), y(t)]^T$ to the original system, define the deviations from equilibrium u(t), v(t) by

$$u(t) := x(t) - x_*$$

$$v(t) := y(t) - y_* .$$

Equivalently,

$$x(t) := x_* + u(t)$$

$$y(t) := y_* + v(t)$$

Thus

$$u' = x' = F(x, y) = F(x_* + u, y_* + v)$$

 $v' = y' = G(x, y) = G(x_* + u, y_* + v)$.

Using partial derivatives, which measure rates of change in the coordinate directions, we can approximate

$$u' = F(x_* + u, y_* + v) = F(x_*, y_*) + \frac{\partial F}{\partial x}(x_*, y_*) u + \frac{\partial F}{\partial y}(x_*, y_*) v + \varepsilon_1(u, v)$$

$$v' = G(x_* + u, y_* + v) = G(x_*, y_*) + \frac{\partial G}{\partial x}(x_*, y_*) u + \frac{\partial G}{\partial y}(x_*, y_*) v + \varepsilon_2(u, v)$$

For differentiable functions, the error terms ε_1 , ε_2 shrink more quickly than the linear terms, as $u, v \rightarrow 0$. Also, note that $F(x_*, y_*) = G(x_*, y_*) = 0$ because (x_*, y_*) is an equilibrium point. Thus the linearized system that approximates the non-linear system for u(t), v(t), is (written in matrix vector form as):

$$\begin{bmatrix} u'(t) \\ v'(t) \end{bmatrix} = \begin{bmatrix} \frac{\partial F}{\partial x}(x_*, y_*) & \frac{\partial F}{\partial y}(x_*, y_*) \\ \frac{\partial G}{\partial x}(x_*, y_*) & \frac{\partial G}{\partial y}(x_*, y_*) \end{bmatrix} \begin{bmatrix} u \\ v \end{bmatrix}.$$

The matrix of partial derivatives is called the <u>Jacobian matrix</u> for the vector-valued function $[F(x, y), G(x, y)]^T$, evaluated at the point (x_*, y_*) . Notice that it is evaluated at the equilibrium point.

People often use the subscript notation for partial derivatives to save writing, e.g. F_x for $\frac{\partial F}{\partial x}$ and F_y for $\frac{\partial F}{\partial y}$.

Example 3) We will linearize the rabbit-squirrel (competition) model of the previous example, near the equilibrium solution $[4, 6]^T$. For convenience, here is that system:

$$x'(t) = 14 x - 2 x^{2} - xy = (x, y)$$

 $y'(t) = 16 y - 2 y^{2} - xy = (x, y)$

 $y'(t) = 16y - 2y^2 - xy = 4 (x)$ 3a) Use the Jacobian matrix method of linearizing they system at $[4, 6]^T$. In other words, as on the previous page, set

$$u(t) = x(t) - 4$$
$$v(t) = y(t) - 6$$

So, u(t), v(t) are the deviations of x(t), y(t) from 4, 6, respectively. Then use the Jacobian matrix computation to verify that the linearized system of differential equations that u(t), v(t) approximately satisfy is

$$\begin{bmatrix} u'(t) \\ v'(t) \end{bmatrix} = \begin{bmatrix} -8 & -4 \\ -6 & -12 \end{bmatrix} \begin{bmatrix} u(t) \\ v(t) \end{bmatrix}.$$

$$J = \begin{bmatrix} \frac{2F}{2x} & \frac{2F}{2y} \\ \frac{2G}{2x} & \frac{2G}{2y} \end{bmatrix} = \begin{bmatrix} 14 - 4x - y & -x \\ -y & 16 - 4y - x \end{bmatrix}$$

$$J \begin{bmatrix} 4 \\ 6 \end{bmatrix} = \begin{bmatrix} -8 & -4 \\ -6 & -12 \end{bmatrix}$$

<u>3b)</u> The matrix in the linear system of DE's above has approximate eigendata:

$$\lambda_1 \approx -4.7, \quad \underline{\mathbf{v}}_1 \approx [.79, -.64]^T$$
 $\lambda_2 \approx -15.3, \quad \underline{\mathbf{v}}_2 \approx [.49, .89]^T$

We can use the eigendata above to write down the general solution to the homogeneous (linearized) system, to make a rough sketch of the solution trajectories to the linearized problem near $[u, v]^T = [0, 0]^T$, and to classify the equilibrium solution using the Chapter 5 cases. Let's do that and then compare our work to the pplane output on the next page. As we'd expect, phase portrait for the linearized problem near $[u, v]^T = [0, 0]^T$ looks very much like the phase portrait for $[x, y]^{\bar{T}}$ near $[4, 6]^T$. This is sensible, since the correspondence between (x, y) and (u, v) involves a translation of x - y coordinate axes to u - vcoordinate axes, via the formula.

Imula.
$$x = u + 4$$

$$y = v + 6$$

$$\begin{bmatrix} 1 \\ 1 \end{bmatrix} = c_1 e^{-15.3} + c_2 e^{-15.3} + c_3 e^{-15.3} + c_4 e^{-15.3} + c_4 e^{-15.3} + c_5 e^{-15.3} + c_5 e^{-15.3} + c_5 e^{-15.3} + c_5 e^{-15.3} + c_6 e^{-15.3} + c$$

Monday: finish Chapter 6.

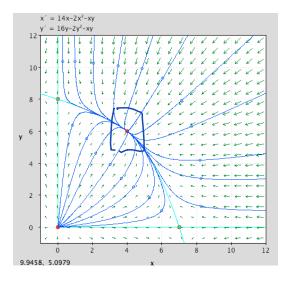
save time for 6.3.182 Hw at and of class (predater-prog)

Wednesday (reading day) 8:05-9:25 go thru old final exam.

(Final exam Foriday)

Linearization allows us to approximate and understand solutions to non-linear problems near equilbria:

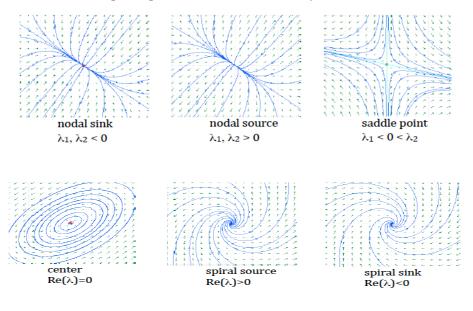
The non-linear problem and representative solution curves:



pplane will do the eigenvalue-eigenvector linearization computation for you, if you use the "find an equilibrium solution" option under the "solution" menu item.

```
Equilbrium Point:
There is a nodal sink at (4, 6)
Jacobian:
-8 -4
-6 -12
The eigenvalues and eigenvectors are:
-4.7085 (0.77218, -0.63541)
-15.292 (0.48097, 0.87674)
```

The solutions to the linearized system near $[u, v]^T = [0, 0]^T$ are close to the exact solutions for non-linear deviations, so under the translation of coordinates $u = x - x_*$, $v = y - y_*$ the phase portrait for the linearized system looks like the phase portrait for the non-linear system.



<u>Theorem:</u> Let $[x_*, y_*]$ be an equilibrium point for a first order autonomous system of differential equations.

- (i) If the linearized system of differential equations at $[x_*, y_*]$ has real eigendata, and either of an (asymptotically stable) nodal sink, an (unstable) nodal source, or an (unstable) saddle point, then the equilibrium solution for the non-linear system inherits the same stability and geometric properties as the linearized solutions.
- (ii) If the linearized system has complex eigendata, and if $\Re(\lambda) \neq 0$, then the equilibrium solution for the non-linear system is also either an (unstable) spiral source or a (stable) spiral sink. If the linearization yields a (stable) center, then further work is needed to deduce stability properties for the nonlinear system.

Example 4 Returning to the non-linear pendulum

$$x'(t) = y = F(x,y)$$

$$y'(t) = -\frac{g}{L}\sin(x) = G(x,y)$$

$$\begin{bmatrix} x \\ y \end{bmatrix} = \begin{bmatrix} h\bar{u} \\ 0 \end{bmatrix}$$

bodenline

The solution trajectories ("orbits") follow level curves of the total energy function, which we repeat from page 1, recalling that $x(t) = \theta(t)$, $y(t) = \theta'(t)$,

 $TE(t) = \frac{1}{2}m(Ly)^2 + mgL(1 - \cos(x))$ TE is strictly minimized at the problem (y = 0) $\cos x = 1$ |y| = 0 |y| = 0

If we compute the Jacobian matrix for this system, we get

$$J(x,y) = \begin{bmatrix} \frac{\partial F}{\partial x} & \frac{\partial F}{\partial y} \\ \frac{\partial G}{\partial x} & \frac{\partial G}{\partial y} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{g}{L}\cos(x) & 0 \end{bmatrix}.$$

• When $x = n\pi$ with n even (and y = 0),

$$J = \begin{bmatrix} 0 & 1 \\ -\frac{g}{L} & 0 \end{bmatrix} \qquad \begin{bmatrix} J - \lambda \overline{J} \end{bmatrix} = \begin{bmatrix} -\lambda & 1 \\ -\frac{1}{L} & -\lambda \end{bmatrix} = \lambda^{2} + \frac{3}{L} = 0$$

the eigenvalues are $\lambda = \pm i \sqrt{\frac{g}{L}}$, so for the linearization we have a stable center, but this is the borderline

case for the non-linear problem. Luckly these equilibrium points are exactly where the total energy function has its strict minimum value (of zero), and if a trajectory starts nearby its total energy is almost zero and the trajectory cannot wander away from the equilibrium point - so these are stable centers for the non-linear pendulum.

• When $x = n\pi$ with n odd (and y = 0),

$$J = \begin{bmatrix} 0 & 1 \\ +\frac{g}{L} & 0 \end{bmatrix} \qquad \begin{bmatrix} J - \lambda I \end{bmatrix} = \begin{bmatrix} -\lambda & 1 \\ \frac{1}{2} & -\lambda \end{bmatrix} = \lambda^2 - \frac{1}{2} = 0$$

the eigenvalues are $\lambda=\pm\sqrt{\frac{g}{L}}$, so for the linearization and the non-linear system we have an unstable saddle!

Example 5) Consider the slightly damped pendulum with $\theta(t)$ with $\frac{g}{L} = 1$ and satisfying

$$\theta''(t) + .2 \theta'(t) + \sin(\theta) = 0$$

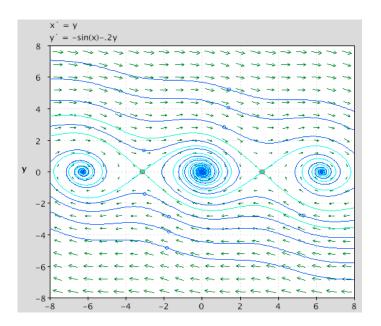
so that $[\theta(t), \theta'(t)]^T$ satisfies

$$x'(t) = y$$

$$y'(t) = -\sin(x) - 0.2y$$

$$x'(t) = -\sin(x) - 0.2y$$
One can check that we get the same equilibrium points as before, corresponding to the pendulum at rest

One can check that we get the same equilibrium points as before, corresponding to the pendulum at rest vertically. The points $(x, y) = (n \pi, 0)$ with n odd are still saddles, but when n is even the stable centers are replaced with spiral sinks. This is an "underdamped" pendulum!



saddle
non-linear
problem
we have
saddle pt
at [nr]

There are lots of interesting population models in section 9.2. Here's another competition model that looks deceptively like Example 2, except the competition got too intense (compare coefficients between the two systems).

Example 6)

$$x'(t) = 14 x - x^2 - 2 x y$$

 $y'(t) = 16 y - y^2 - 2 x y$

Do populations peacefully co-exist in this competition model? A little competition may be healthy, but too much maybe not so much. :-)

competition models $x' = a_1 x - b_1 x^2 - c_1 xy$ $y' = a_2 x - b_2 x^2 - c_2 xy$ in embien example, peaceful erexistence 16 14 Fact: (in this example bib2 = 1 cic2 = 4 in ember example bib2 = 4 cic2 = 1 if competition model

has an equil. in $|S^{t}|$ quadrant $\begin{bmatrix} x_{u} \\ y_{v} \end{bmatrix} \text{ with } x_{u}, y_{u} > 0$ $\begin{cases} x_{u} \\ y_{v} \end{bmatrix} \text{ with } x_{u}, y_{u} > 0$ $\begin{cases} x_{u} \\ y_{v} \end{bmatrix} \text{ is stable } \lambda \text{ all } \begin{bmatrix} x_{u}(t) \\ y_{u}(t) \end{bmatrix} \rightarrow \begin{bmatrix} x_{u} \\ y_{v} \end{bmatrix} \text{ if } c_{1}(c_{1}) b_{1}b_{2} \text{ then } \begin{bmatrix} x_{u} \\ y_{v} \end{bmatrix} \text{ is unstable } \begin{pmatrix} x_{u} \\ y_{v} \end{pmatrix} \text{ one population dries out}$ Predator-Prey: x(t) is the prey, y(t) is the predator. One model:

$$x'(t) = a x - p x y = x(a - py)$$

 $y'(t) = -b y + q x y = y(-b + q x)$

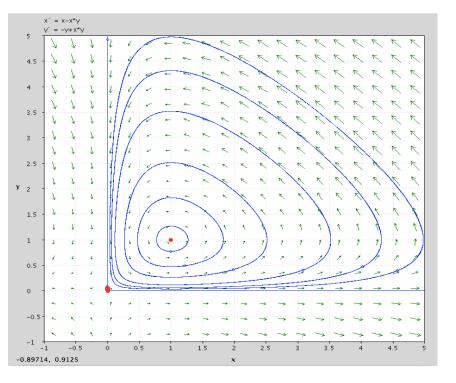
natural region of interest is the first quadrant. Equilibrium solutions (0,0), $\left(\frac{b}{q},\frac{a}{p}\right)$.

• Linearize at $\left(\frac{b}{q}, \frac{a}{p}\right)$ gives a stable center, need to do more work to deduce whether this equilibrium solution is a stable center for the non-linear system. (It is.) Example:

ample.
$$x'(t) = x - xy = x(1 - y) = F(x, y)$$

$$y'(t) = -y + xy = y(-1 + x) = G(x, y)$$

$$= \begin{bmatrix} \frac{\partial F}{\partial x} & \frac{\partial F}{\partial y} \\ \frac{\partial G}{\partial x} & \frac{\partial G}{\partial y} \end{bmatrix} = \begin{bmatrix} 1 - y & -x \\ y & -1 + x \end{bmatrix} \quad \text{again } S \text{ this } \begin{bmatrix} 0 \\ 0 \end{bmatrix}, \begin{bmatrix} 1 \\ 1 \end{bmatrix}.$$



$$\begin{bmatrix}
0 & [0] \\
0 & -1
\end{bmatrix}$$
Saddle
$$\begin{bmatrix}
0 & [1] \\
0 & -1
\end{bmatrix}$$

$$\begin{bmatrix}
-1 & [1 & -1] \\
1 & [1 & -\lambda]
\end{bmatrix}$$

$$\begin{bmatrix}
-1 & [1 & -\lambda] \\
-1 & [1 & -\lambda]
\end{bmatrix}$$

$$\begin{bmatrix}
-1 & [1 & -\lambda] \\
-1 & [1 & -\lambda]
\end{bmatrix}$$

$$\begin{bmatrix}
-1 & [1 & -\lambda] \\
-1 & [1 & -\lambda]
\end{bmatrix}$$

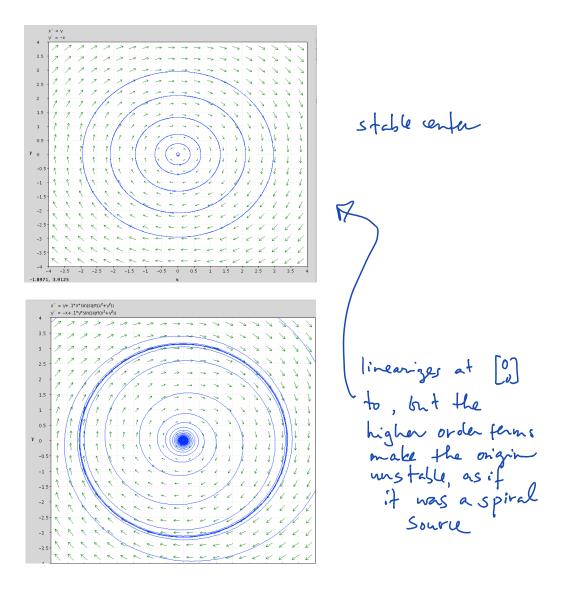
$$\begin{bmatrix}
-1 & [1 & -\lambda] \\
-1 & [1 & -\lambda]
\end{bmatrix}$$

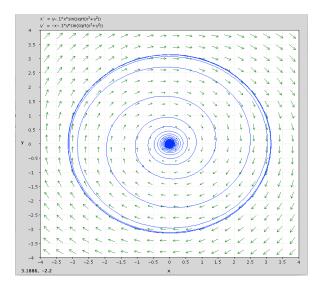
$$\begin{bmatrix}
-1 & [1 & -\lambda] \\
-1 & [1 & -\lambda]
\end{bmatrix}$$

$$\begin{bmatrix}
-1 & [1 & -\lambda] \\
-1 & [1 & -\lambda]
\end{bmatrix}$$

for lineary ation
borderline for non-linear,
but using separable DE's
you can show ['] is stable

Stable center for linearization is borderline for nonlinear problem:





also linearizes at origin to stable center. In this case the higher order terms make origin behave like a spiral sink.