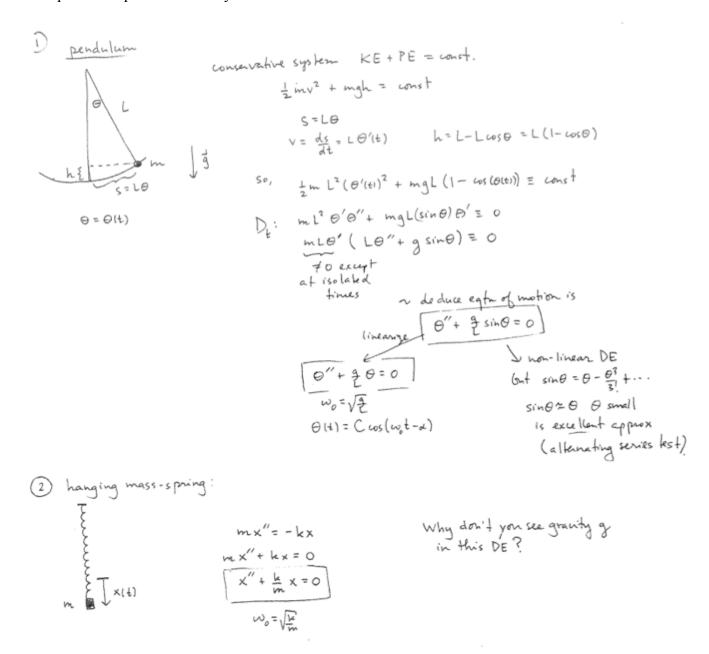
Math 2250-4 Fri Nov 1 Experiment day :-) Superquiz day :-/

Small oscillation pendulum motion and vertical mass-spring motion are governed by exactly the "same" differential equation that models the motion of the mass in a horizontal mass-spring configuration. The nicest derivation for the pendulum depends on conservation of mass, as indicated below. Today we will test both models with actual experiments (in the undamped cases), to see if the predicted periods $T = \frac{2\pi}{\omega_0}$ correspond to experimental reality.



Pendulum: measurements and prediction (we'll check these numbers).

>
$$L := 1.526$$
;
 $g := 9.806$;
 $\omega := \sqrt{\frac{g}{L}}$; # radians per second
 $f := evalf(\omega/(2 \cdot Pi))$; # cycles per second
 $T := 1/f$, # seconds per cycle

Experiment:

Mass-spring:

compute Hooke's constant:

[> 98.7 - 83.4; #displacement from extra 50g]

>
$$k := \frac{.05 \cdot 9.806}{.153}$$
; # solve $k \cdot x = m \cdot g$ for k .

> $m := .1$; # mass for experiment is 100g

$$\omega := \sqrt{\frac{k}{m}}$$
; # predicted angular frequency

 $f := evalf\left(\frac{\omega}{2 \cdot Pi}\right)$; # predicted frequency

 $T := \frac{1}{f}$; # predicted period

Experiment:

Experiment.

We neglected the KE_{spring} , which is small but could be adding intertia to the system and slowing down the oscillations. We can account for this:

Improved mass-spring model

Normalize TE = KE + PE = 0 for mass hanging in equilibrium position, at rest. Then for system in motion,

$$KE + PE = KE_{mass} + KE_{spring} + PE_{work}.$$

$$PE_{work} = \int_{0}^{x} k s \, ds = \frac{1}{2} k x^{2}, \quad KE_{mass} = \frac{1}{2} m \left(x'(t) \right)^{2}, \quad KE_{spring} = ????$$

How to model KE_{spring} ? Spring is at rest at top (where it's attached to bar), moving with velocity x'(t) at bottom (where it's attached to mass). Assume it's moving with velocity $\mu x'(t)$ at location which is fraction μ of the way from the top to the mass. Then we can compute KE_{spring} as an integral with respect to μ , as the fraction varies $0 \le \mu \le 1$:

$$KE_{spring} = \int_{0}^{1} \frac{1}{2} (\mu x'(t))^{2} (m_{spring} d\mu)$$

$$= \frac{1}{2} m_{spring} (x'(t))^{2} \int_{0}^{1} \mu^{2} d\mu = \frac{1}{6} m_{spring} (x'(t))^{2}.$$

Thus

$$TE = \frac{1}{2} \left(m + \frac{1}{3} m_{spring} \right) (x'(t))^2 + \frac{1}{2} k x^2 = \frac{1}{2} M(x'(t))^2 + \frac{1}{2} k x^2,$$

where

$$M = m + \frac{1}{3} m_{spring}$$

$$D_t(TE) = 0 \Rightarrow$$

$$Mx'(t) x''(t) + kx(t) x'(t) = 0$$
.
 $x'(t) (Mx'' + kx) = 0$.

Since x'(t) = 0 only at isolated t-values, we deduce that the corrected equation of motion is

(Mx'' + kx) = 0

with

$$\omega_0 = \sqrt{\frac{k}{M}} = \sqrt{\frac{k}{m + \frac{1}{3} m_{spring}}} .$$

Does this lead to a better comparison between model and experiment?

>
$$ms := .011$$
; # spring has mass 11g
$$M := m + \frac{1}{3} \cdot ms$$
; # "effective mass"

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$$M := m + \frac{1}{3} \cdot ms$$
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> $\omega := \sqrt{\frac{k}{M}}$; # $predicted$ angular frequency
$$f := evalf\left(\frac{\omega}{2 \cdot Pi}\right)$$
; # $predicted$ frequency
$$T := \frac{1}{f}$$
; # $predicted$ period