

Math 2250-1  
 Tuesday Oct 28

①

§5.4 cont'd.

We are studying unforced mechanical vibrations

$$m x'' + c x' + k x = 0$$

On Monday we studied

Case 1 Free undamped motion ( $c=0$ )

$$m x'' + k x = 0$$

$$x'' + \omega_0^2 x = 0 \quad \omega_0 = \sqrt{\frac{k}{m}}$$

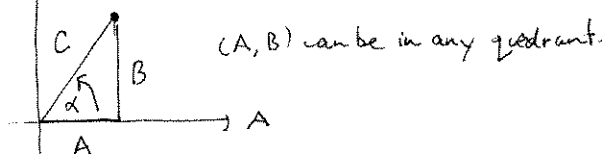
sol'n

$$x(t) = A \cos \omega_0 t + B \sin \omega_0 t$$

$$= C \cos(\omega_0 t - \alpha)$$

$B \uparrow$   
 amplitude  
 $\uparrow$   
 phase

(review; we won't go over this in class)



$$C = \text{amplitude} = \sqrt{A^2 + B^2}$$

$\omega_0 = \text{angular freq (rad/time)}$

$f = \frac{\omega_0}{2\pi}$  (cycles/sec (hertz)) frequency

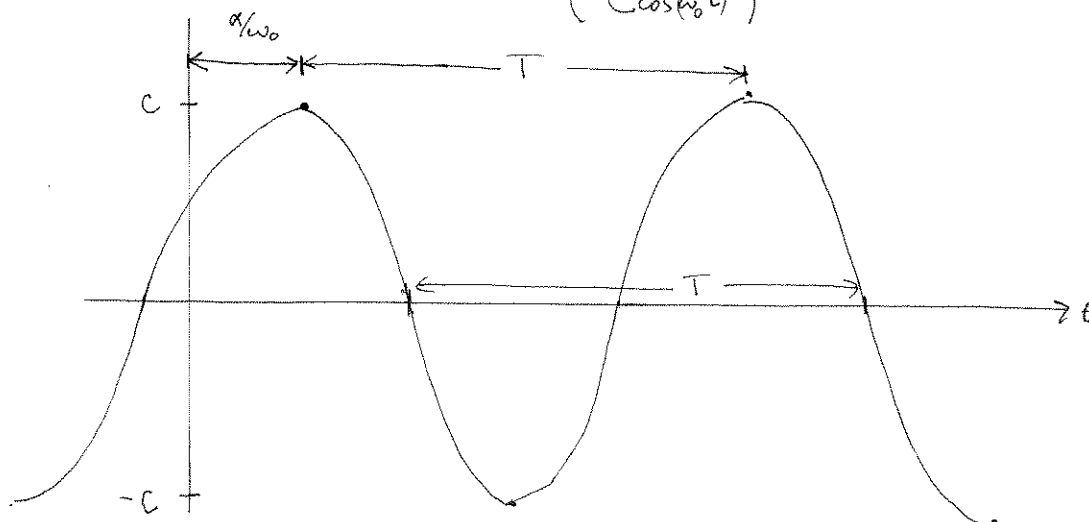
$T = \frac{2\pi}{\omega_0}$  (time/cycles) period

$\alpha = \text{phase angle;}$

since  $\omega_0 t - \alpha = \omega_0 (t - \frac{\alpha}{\omega_0})$   $\delta = \frac{\alpha}{\omega_0}$  is the time delay

this shifts standard cos curve to the right by  $\frac{\alpha}{\omega_0}$

( $C \cos(\omega_0 t)$ )



Case 2: Damping

$$m x'' + c x' + k x = 0$$

$$x'' + 2p x' + \omega_0^2 x = 0$$

$$\omega_0 = \sqrt{\frac{k}{m}} \text{ still}; \quad \frac{c}{m} = 2p; \quad p = \frac{c}{2m}$$

$$p(r) = r^2 + 2pr + \omega_0^2 = 0$$

$$r = \frac{-2p \pm \sqrt{4p^2 - 4\omega_0^2}}{2}$$

$$r = -p \pm \sqrt{p^2 - \omega_0^2}$$

Overdamped:  $p^2 - \omega_0^2 > 0$  ( $c^2 > 4km$ )

$$r = r_1, r_2 < 0$$

$$x(t) = c_1 e^{r_1 t} + c_2 e^{r_2 t} = e^{r_1 t} (c_1 + c_2 e^{(r_2 - r_1)t})$$

Figure 5.4.7 p 327

→ sol's decay exponentially to zero, cross t-axis at most 1 time.

Critically damped  $p^2 - \omega_0^2 = 0$  ( $c^2 = 4km$ )

$$r = -p \text{ double root}$$

$$x(t) = c_1 e^{rt} + c_2 t e^{rt} = e^{rt} (c_1 + c_2 t)$$

Figure 5.4.8 p 327 → sol's decay exponentially to zero, cross t-axis at most once

Underdamped  $p^2 - \omega_0^2 < 0$

$$\text{set } \omega_1 = \sqrt{\omega_0^2 - p^2} < \omega_0$$

$$r = -p \pm i\omega_1$$

$$e^{rt} = e^{(-p \pm i\omega)t}$$

$$x(t) = e^{-pt} (A \cos \omega_1 t + B \sin \omega_1 t)$$

$$= e^{-pt} (C \cos(\omega_1 t - \alpha))$$

Figure 5.4.9 p 328

$$\text{pseudo period } T = \frac{2\pi}{\omega_1}$$

$$\text{pseudo freq } f = \frac{\omega_1}{2\pi}$$

$$\text{pseudo ang. freq } \omega_1$$

$C e^{-pt}$  : " time varying amplitude

① solution oscillates, but oscillations are damped exponentially

② motion is slowed relative to no damping ( $\omega_1 < \omega_0$ ).

Exercise 1 On Monday we solved an undamped problem  $c=0, m=2, k=18$

$$\begin{cases} x'' + 9x = 0 \\ x(0) = 1 \\ x'(0) = \frac{3}{2} \end{cases}$$

$$\begin{aligned} x_H(t) &= A \cos 3t + B \sin 3t \\ &= \cos 3t + .5 \sin 3t \\ &= \frac{\sqrt{5}}{2} \cos(3t - .46) \quad .46 = \arctan\left(\frac{.5}{1}\right) \\ &= \frac{\sqrt{5}}{2} \cos(3(t - .15)) \\ T &= \frac{2\pi}{3} \approx 2.09 \end{aligned}$$

Now add damping!

1a)  $m=2, k=18, c=4$ , same IV's

$$\begin{cases} x'' + 2x' + 9x = 0 \\ x(0) = 1 \\ x'(0) = \frac{3}{2} \end{cases}$$

find  $x_H(t)$ , general sol'n

Solve IVP

type of damping?

discuss

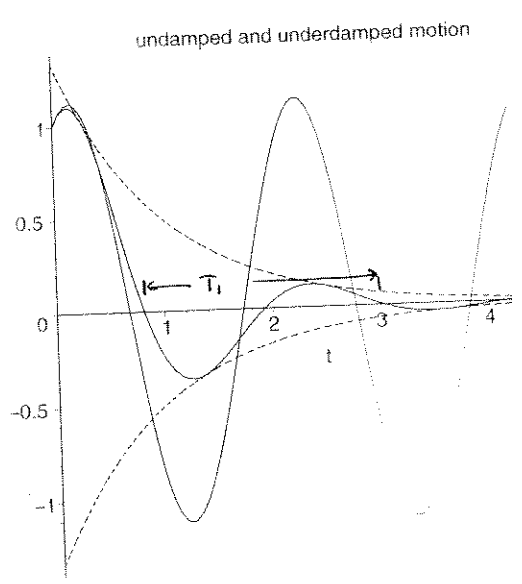
underdamped example

```
> restart;
> with(DEtools):
> Digits:=5:
> deqtn1:=diff(x(t),t,t)+2*diff(x(t),t)+9*x(t)=0:
> IC:=x(0)=1,D(x)(0)=1.5:
> dsolve({deqtn1,IC},x(t));

x(t) = 5/8*sqrt(2)*e^(-t)*sin(2*sqrt(2)*t) + e^(-t)*cos(2*sqrt(2)*t)

> evalf(%);

x(t) = 0.88388 e^(-1.0 t) sin(2.8284 t) + e^(-1.0 t) cos(2.8284 t)
> x1:=t->.88388*exp(-1.*t)*sin(2.8284*t)+exp(-1.*t)*cos(2.8284*t):
> C1:=sqrt(.88388^2+1); #pseudoamplitude
omega1:=2.8284: #slower (pseudo)-angular
#frequency than w/o damping
T1:=evalf(2*Pi/omega1); #longer "pseudoperiod"
alpha:=arctan(.88388); #pseudophase
C1:=1.3346
T1:=2.2215
alpha:=0.72384
> x2:=t->exp(-t)*C1*cos(omega1*t-alpha): #same function
> with(plots):
> plot1:=plot(x1(t),t=0..5,color=black):
> plot2:=plot(x2(t),t=0..5,color=black):
> plot0:=plot(cos(3*t)+.5*sin(3*t),t=0..5,color=black):
#undamped IVP solution
plot3:=plot(C1*exp(-t),t=0..5,linestyle=2,color=black):
plot4:=plot(-C1*exp(-t),t=0..5,linestyle=2,color=black):
#the exponential "envelope", linestyle=2 means "dot"
display({plot1,plot2,plot0,plot3,plot4},title='undamped and underdamped motion');
```



$$1b) \begin{cases} x'' + 6x' + 9x = 0 \\ x(0) = 1 \\ x'(0) = 3/2 \end{cases}$$

Solve and discuss

$$1c) \begin{cases} x'' + 10x' + 9x = 0 \\ x(0) = 1 \\ x'(0) = 3/2 \end{cases}$$

Solve & discuss

```
> deqtn2:=diff(x(t),t,t)+6*diff(x(t),t)+9*x(t)=0:
IC:=x(0)=1,D(x)(0)=1.5:
dsolve({deqtn2,IC},x(t)); #critical damping
```

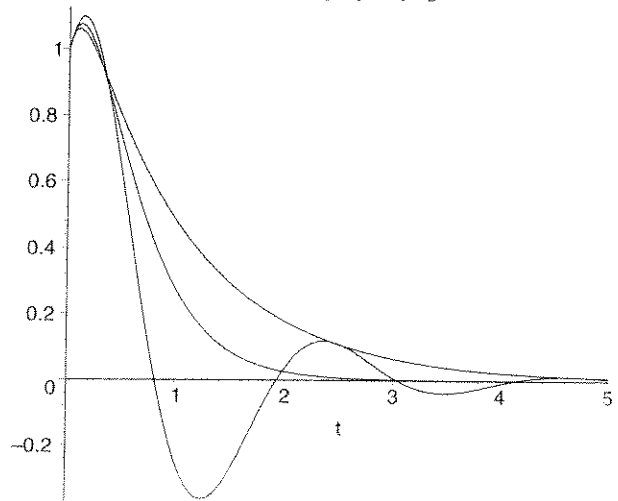
$$x(t) = e^{-3t} + \frac{9}{2} e^{-3t} t$$

```
> deqtn3:=diff(x(t),t,t)+10*diff(x(t),t)+9*x(t)=0:
IC:=x(0)=1,D(x)(0)=1.5:
dsolve({deqtn3,IC},x(t)); #overdamping
```

$$x(t) = \frac{21}{16} e^{-t} - \frac{5}{16} e^{-9t}$$

```
> plot5:=plot(exp(-3*t)+9/2*exp(-3*t)*t,t=0..5,color=black):
plot6:=plot(21/16*exp(-t)-5/16*exp(-9*t),t=0..5,color=black):
display({plot5,plot6,plot2},title='under,critical,and
over-damping, by varying c');
```

under,critical,and  
over-damping, by varying c



So much for unforced (homogeneous) spring problems. Now discuss general non-homogeneous linear DE, then in §5.6 we'll return to springs:

§5.5: Finding  $y_p$ 's for  $\mathcal{L}(y) = f$ .

Recall, if  $\mathcal{L}$  is a linear operator the general sol'n to

$$\mathcal{L}(y) = f$$

is  $y = y_p + y_H$

where  $y_p$  is any particular sol'n and  $y_H$  is the general (n-dim'l) solution to  $\mathcal{L}(y) = 0$

How to find  $y_p$ 's.

Book calls the method "undetermined coefficients", for constant coefficient linear DE's, is really just "guessing" (but could be justified with vector space theory).

example 1

$$\mathcal{L}(y) = y'' + 3y' + 4y$$

solve  $\mathcal{L}(y) = 3x + 2$

try  $y_p = Ax + B$  (because when you apply  $\mathcal{L}$  to polys of degree  $\leq 1$  you get back polys of degree  $\leq 1$ ).

$$\begin{aligned} 4 (y_p &= Ax + B) \\ + 3 (y_p' &= A) \\ + 1 (y_p'' &= 0) \end{aligned}$$

$$\mathcal{L}(y_p) = x(4A) + 1(3A + 4B) = 3x + 2$$

$$\left. \begin{aligned} 4A &= 3 \\ 3A + 4B &= 2 \end{aligned} \right\} \begin{aligned} A &= \frac{3}{4} \\ \frac{9}{4} + 4B &= 2 \\ 4B &= -\frac{1}{4} \\ B &= -\frac{1}{16} \end{aligned}$$

$$y_p = \frac{3}{4}x - \frac{1}{16}$$

$$y_H: r^2 + 3r + 4 = 0$$

$$\begin{aligned} r &= \frac{-3 \pm \sqrt{9-16}}{2} \\ &= \frac{-3 \pm i\sqrt{7}}{2} \end{aligned}$$

$$y_H(x) = e^{-3/2 x} (A \cos \frac{\sqrt{7}}{2} x + B \sin \frac{\sqrt{7}}{2} x)$$

$$y(x) = \frac{3}{4}x - \frac{1}{16} + e^{-3/2 x} (A \cos \frac{\sqrt{7}}{2} x + B \sin \frac{\sqrt{7}}{2} x)$$

Tomorrow, we'll continue the "method of guessing" for right-hand sides of certain types. See table p.341 (poly, exps, trig, exp-trig)